

***Generalized shape optimization with X-FEM
and Level Set description applied to stress
constrained structures***

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Outline

- Introduction
- eXtended Finite Element Method (XFEM)
- Level Set Method
- Sensitivity Analysis
- Applications
- Conclusion

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INTRODUCTION

XFEM

LEVEL SET

SENSITIVITY

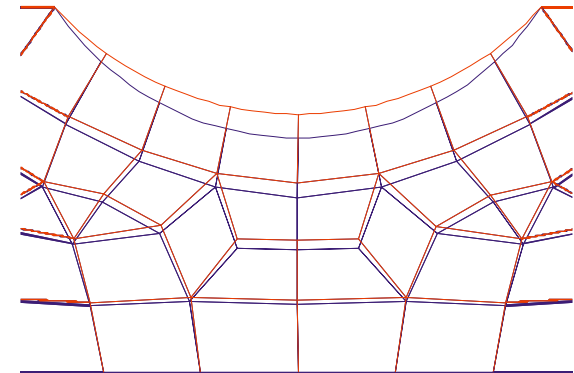
APPLICATIONS

CONCLUSION

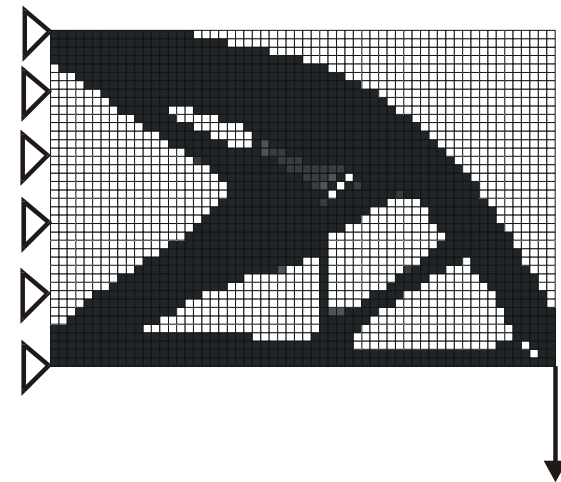
Introduction – Optimization of mechanical structures



- Shape optimization :
 - Variables : geometric parameters
 - Limitation on shape modification
 - Mesh perturbation – Velocity field
 - CAD model of structure
 - Various formulations available



- Topology optimization :
 - Variables : distribution of material element densities
 - Strong performance – deep change allowed
 - Fixed Mesh
 - « Image » of the structure
 - Limitation of objective function and constraints



Introduction – X-FEM and Level Set optimization



- XFEM + Level Set description = **Intermediate** approach between shape and topology optimisation
 - eXtended Finite Element Method → Alternative to remeshing
 - work on fixed mesh
 - **no velocity field** and no mesh perturbation required
 - Level Set Method → Alternative description to parametric CAD
 - Smooth curve description
 - Topology can be altered as entities can be merged or separated → **generalized shape**
 - Problem formulation:
 - Global and local constraints
 - Small number of design variables
- Introduction of new holes requires a topological derivatives

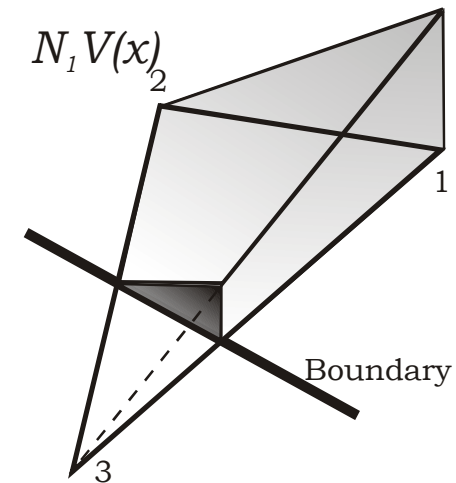
eXtended Finite Element Method



- Early motivation :
 - Study of propagating crack in mechanical structures → avoid the remeshing procedure
- Principle :
 - Allow the model to handle discontinuities that are non conforming with the mesh
- Representing holes or material – void interfaces
 - Use special shape functions $V(x)N_i(x)$ (discontinuous)

$$u = \sum_{i \in I} N_i(x) V(x) u_i \Rightarrow K_{uu} u = f$$

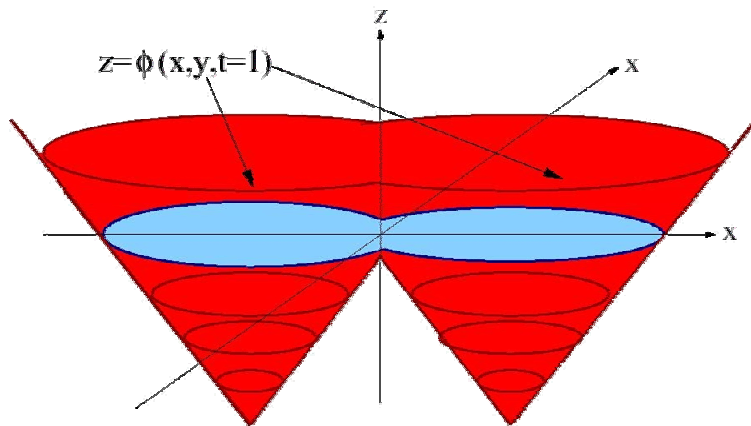
$$V(x) = \begin{cases} 1 & \text{if node} \in \text{solid} \\ 0 & \text{if node} \in \text{void} \end{cases}$$



The Level Set Description



- Principle (Sethian & Osher, 1999)
 - Introduce a higher dimension
 - Implicit representation
 - Interface = the zero level of a function : $\psi(x, t) = 0$
- Possible practical implementation:
 - Approximated on a fixed mesh by the signed distance function to curve Γ :

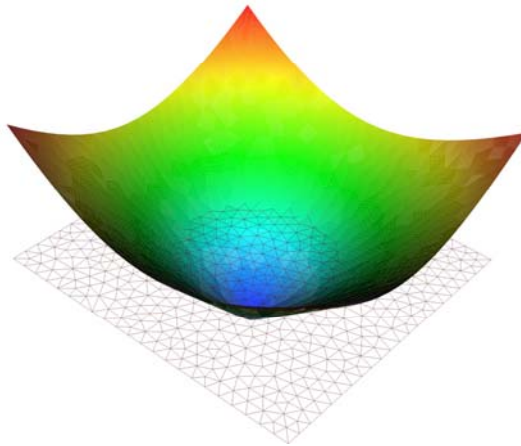


$$\psi(x, t) = \pm \min_{x_\Gamma \in \Gamma(t)} \|x - x_\Gamma\|$$

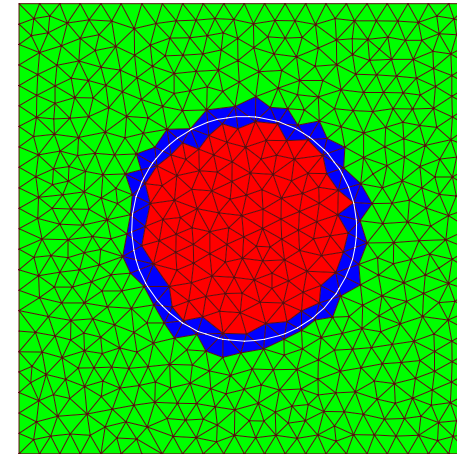
- Advantages:
 - 2D / 3D
 - Combination of entities:
e.g. min / max

The Level Set and the X-FEM

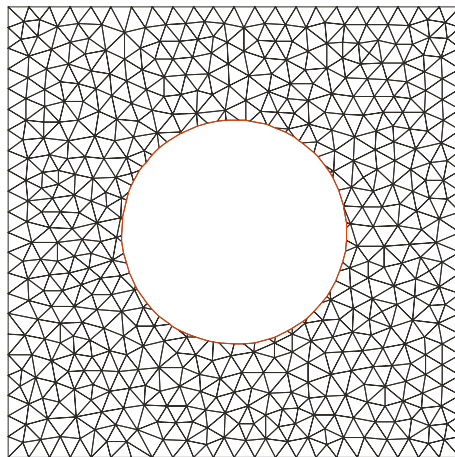
■ 1. Build Level Set



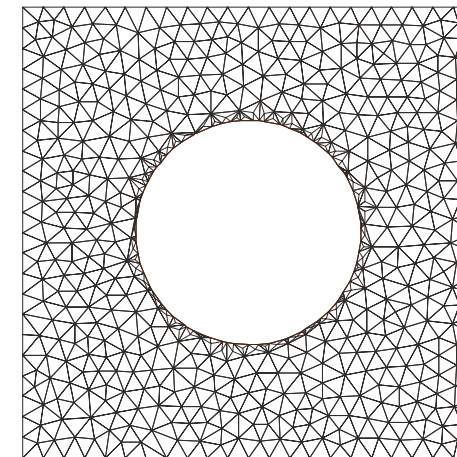
■ 2. Detect type of element



■ 3. Cut the mesh



■ 4. Working mesh for integration



Optimization Problem Formulation



- Geometry description and material layout:
 - Using Basic Level Sets description : ellipses, rectangles, NURBS,...
- Design Problem:
 - Find the best shape to minimize a given objective functions while satisfying design constraints
- Design variables:
 - Parameters of Level Sets
- Objective and constraints:
 - Mechanical responses: compliance, displacement, stress,...
 - Geometrical characteristics: volume, distance,...
- Problem formulation similar to shape optimization but simplified thanks to XFEM and Level Set!

Sensitivity analysis - principles

- Classical approach for sensitivity analysis in industrial codes: **semi analytical** approach based on K derivative

- Discretized equilibrium: $K u = f$

- Derivatives of displacement: $K \frac{\partial u}{\partial x} = \left(\frac{\partial f}{\partial x} - \frac{\partial K}{\partial x} u \right)$

- Semi analytical approach:

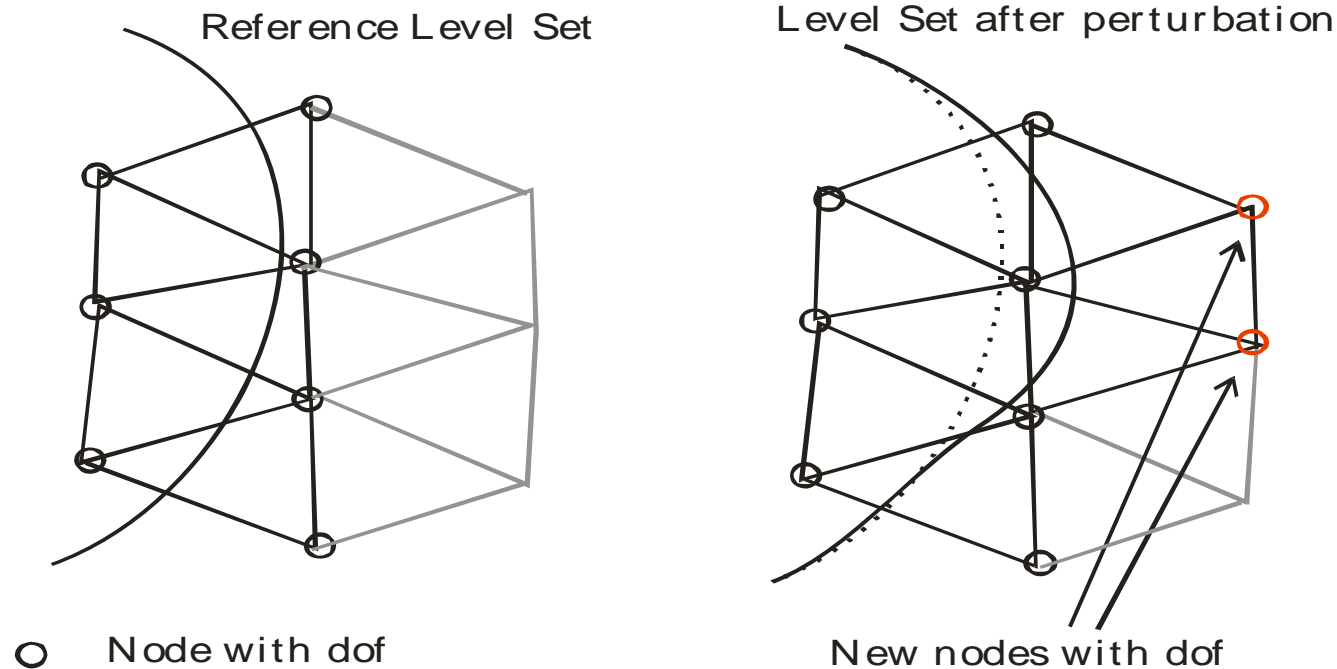
- Stiffness matrix derivative: $\frac{\partial K}{\partial x} \approx \frac{K(x+\delta x) - K(x)}{\delta x}$

- Compliance derivative: $\frac{\partial C}{\partial x} = -u^T \frac{\partial K}{\partial x} u$

- Stress derivative: $\frac{\partial \sigma}{\partial x} = \frac{\sigma(u(x+\delta x)) - \sigma(u(x))}{\delta x}$

Sensitivity analysis – technical difficulty

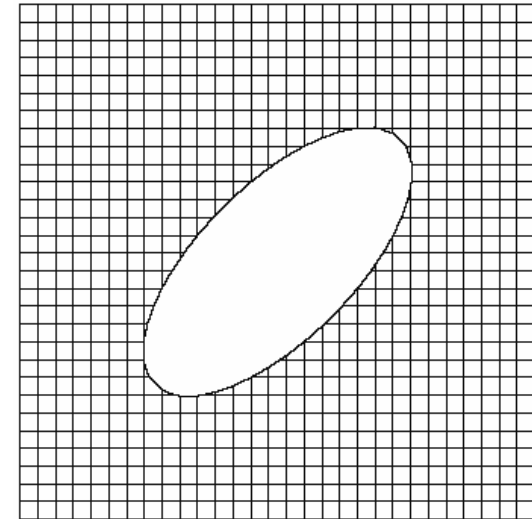
- Introduction of new dofs → K dimension changes → not possible



- Ignore the new elements that become solid or partly solid
 - Small errors, but minor contributions
 - Practically, no problem observed
 - Efficiency and simplicity
 - Validated on benchmarks

Sensitivity analysis – validation

- Validation of semi-analytic sensitivity:
 - Elliptical hole
 - Parameters: major axis a and Orientation angle θ w.r. to horizontal axis
 - Perturbation: $\delta=10^{-4}$



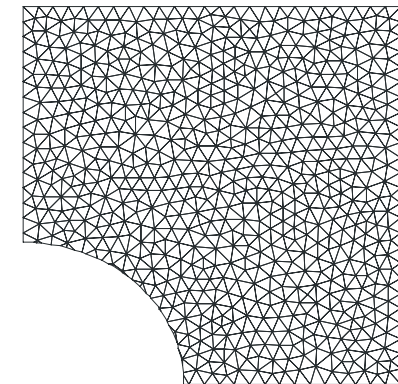
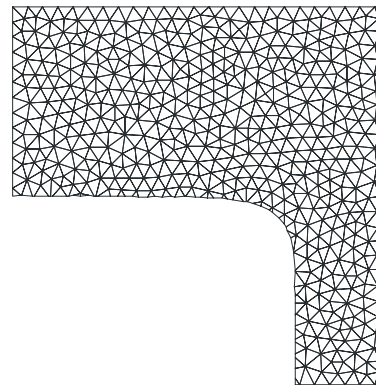
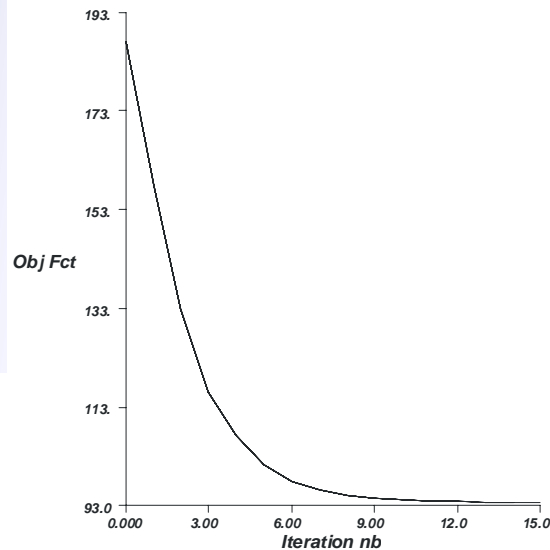
<i>Design variables</i>	<i>Finite differences</i>	<i>Semi-analytical approach</i>	<i>Relative error (%)</i>
$a = 0.6$	3698, 0000	3691, 3344	0, 1802
$\theta = \pi/4$	478, 0000	477, 0641	0, 1957
$a = 0.6$	2712, 000	2707, 328	0, 1722
$\theta = \pi/6$	523, 70000	523, 4099	0, 0553
$a = 0.6$	783, 8000	781, 3920	0, 3072
$\theta = 0$	11, 6239	11, 6235	0, 0029

Applications - 2D plate with a hole

■ Plate with generalized super elliptical hole :

- Parameters : $2 < a, b, \eta, \alpha < 8$
- Objective: min Compliance.
- Constraint: upper bound on the Volume.
- Bi-axial Load: $\sigma_x = \sigma_y = \sigma_0$
- Solution: perfect circle: $a = b = 2, \eta = \alpha = 2$

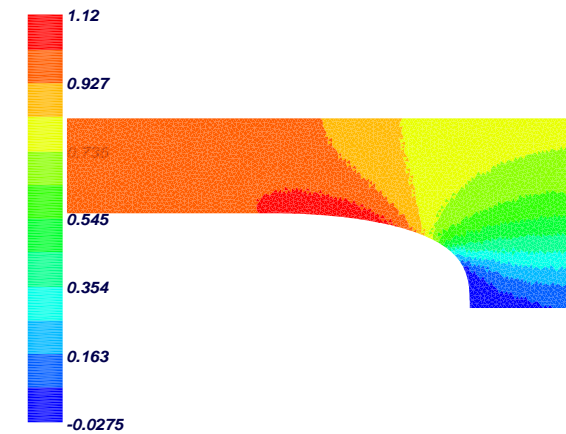
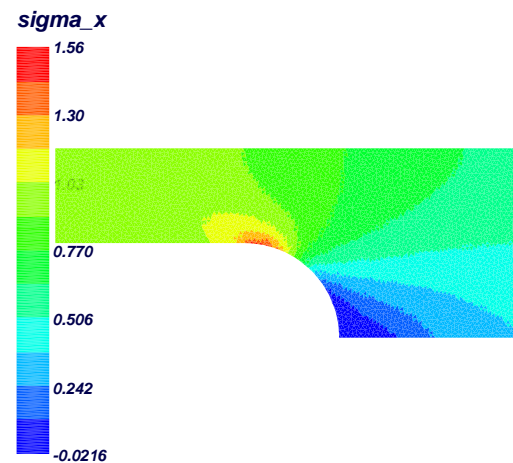
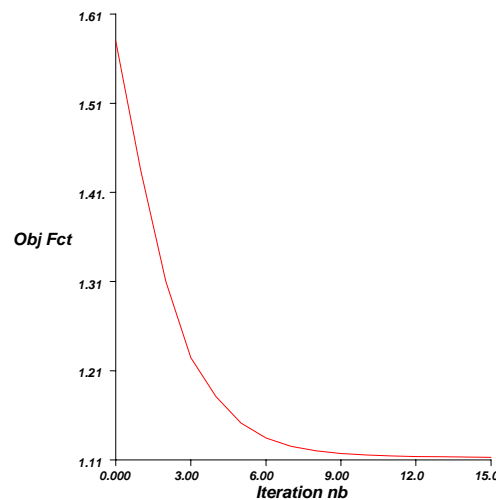
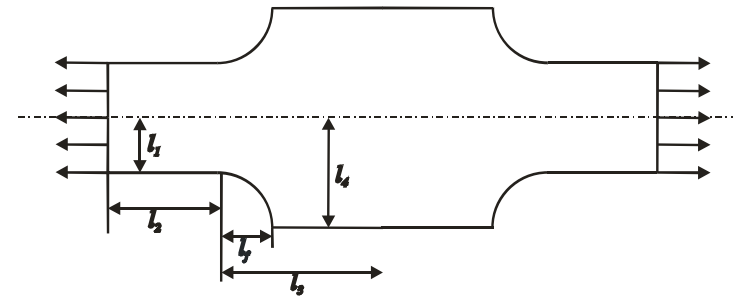
$$\frac{|x|^\alpha}{|a|} + \frac{|y|^\eta}{|b|} = r$$



Applications – 2D fillet in tension

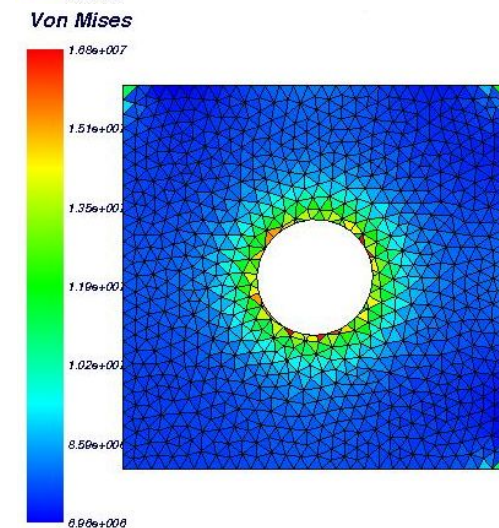
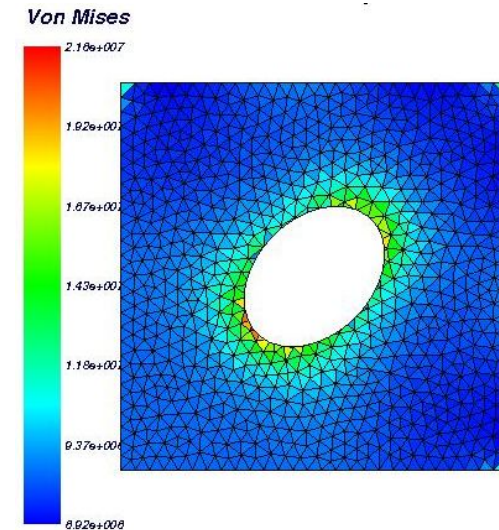
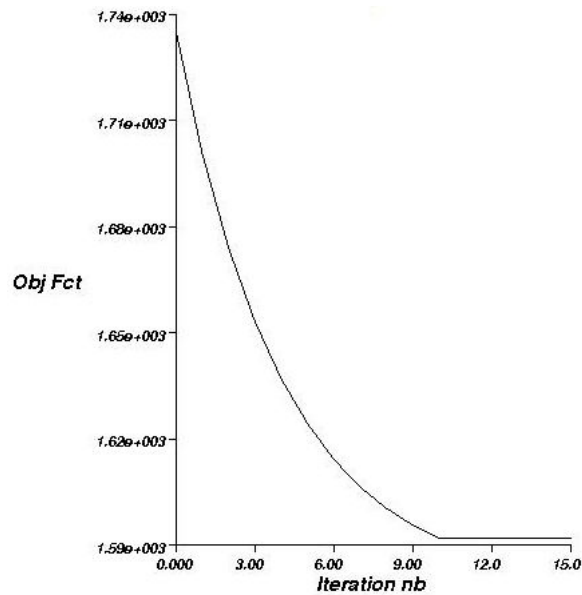
- Shape of the fillet : generalized super ellipse
$$\frac{|x|^\alpha}{|a|} + \frac{|y|^\eta}{|b|} = r$$

- Parameters : a, η, α
- Objective: min (max Stress)
- No Constraint
- Uni-axial Load: $\sigma_x = \sigma_0$
- Solution: stress reduction of 30%



Applications – 2D plate with elliptical hole

- Plate with elliptical hole :
 - Parameters : a, θ
 - Objective: min Compliance.
 - Constraint: $\sigma_{VM} \leq 0.8 \sigma_{VM0}$
 - Bi-axial Load: $\sigma_x = \sigma_y = \sigma_0$
 - Solution: perfect circle. $\sigma_{VM0} : 2.16$
 $\sigma_{VM \text{ final}} : 1.68$





Conclusion

- XFEM and Level Set gives rise to a generalized shape optimisation technique
- Intermediate to shape and topology optimisation
 - Work on a fixed mesh
 - Topology can be modified:
 - Holes can merge and disappear
 - New holes cannot be introduced without topological derivatives
 - Smooth curves description
 - Void-solid description
 - Small number of design variables
 - Global or local response constraints
 - No velocity field and mesh perturbation problems



Conclusion

- Contribution of this work
 - New perspectives of XFEM and Level Set
 - Investigation of semi-analytical approach for sensitivity analysis
 - Implementation in a general C++ multiphysics FE code
- Perspectives:
 - 3D problems
 - Dynamic problems
 - Multiphysic simulation problems with free interfaces